

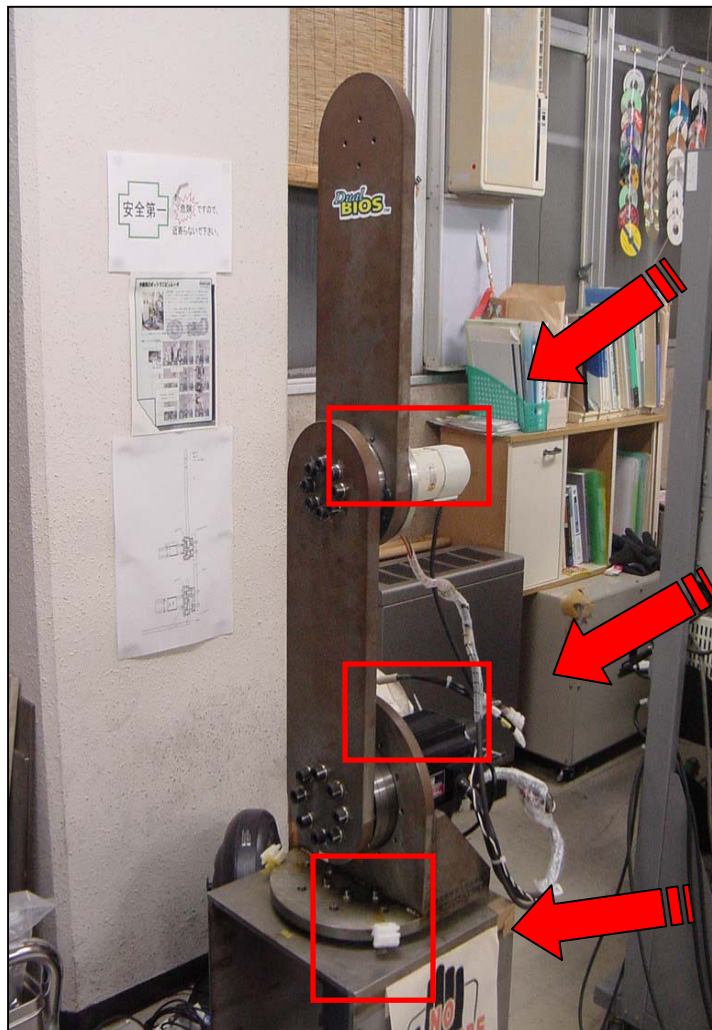


***Robust Motion Control of Industrial Robot
Considering Two-inertia System
Based on CDM Method and Resonance Ratio Control***

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Background

What's need for industrial robot manipulators



High-accuracy

High-repeatability

High-precision

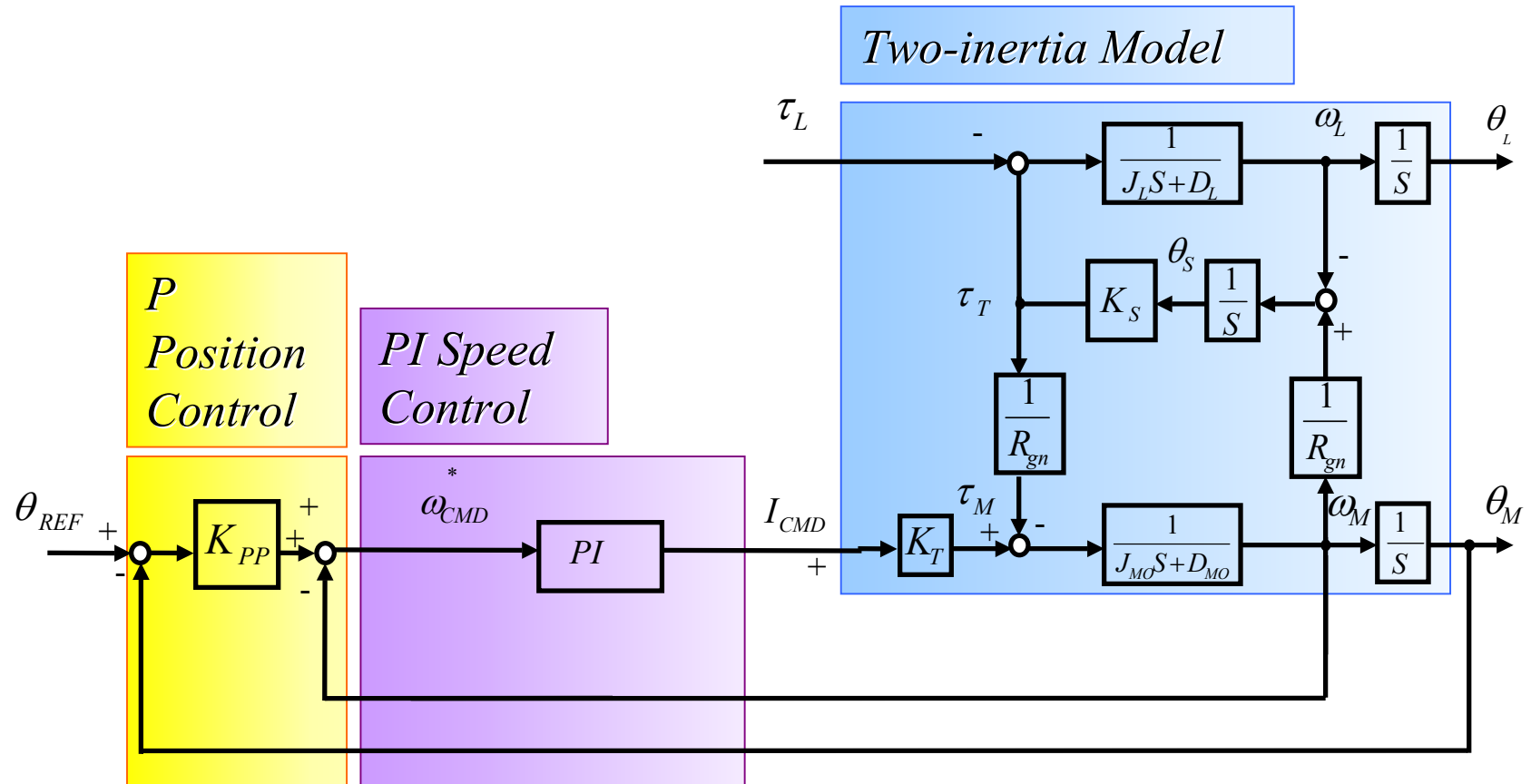
Problem

1. Vibration Suppression

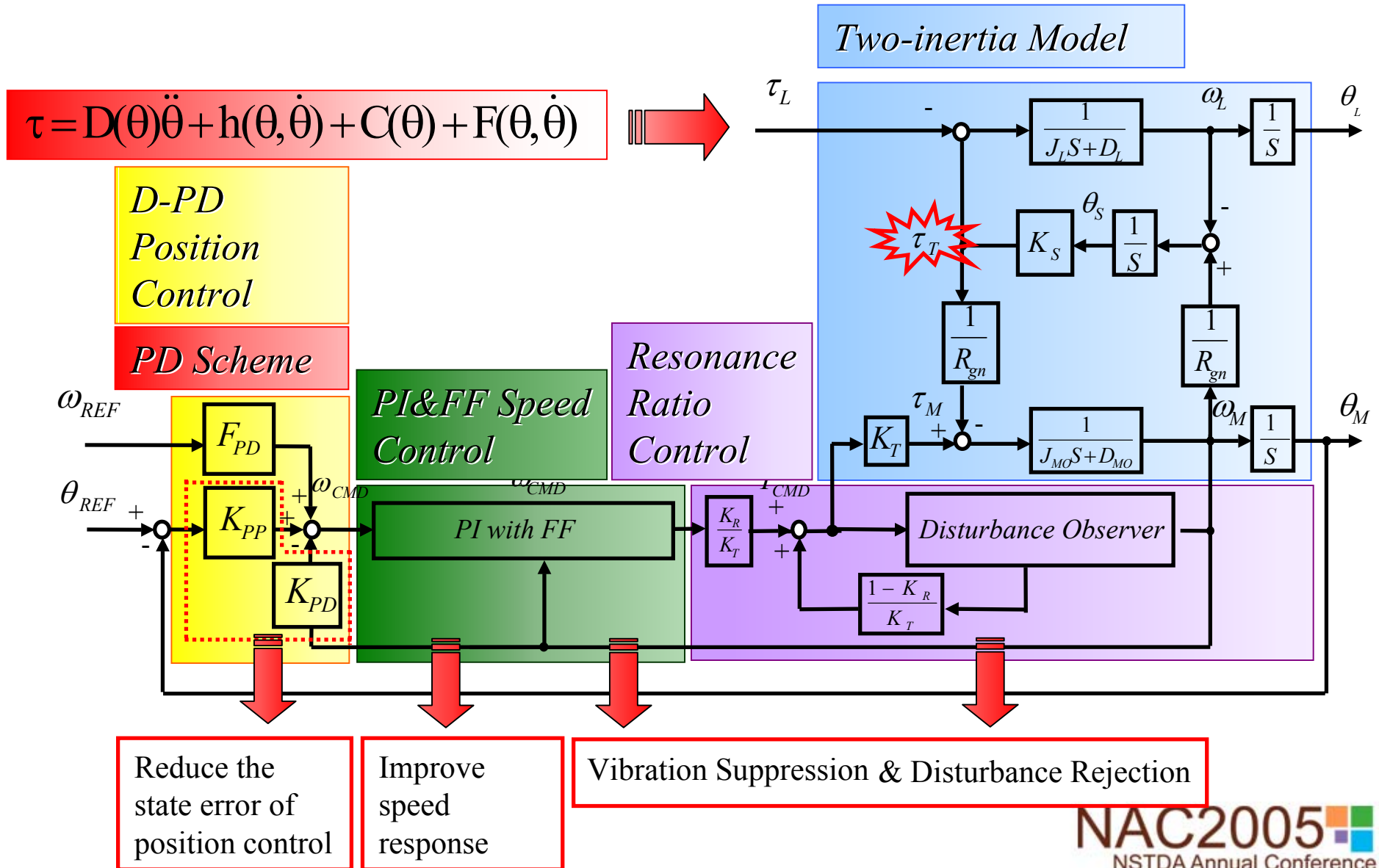
2. Improve speed response

3. Reduce the state error of position control

Outline of conventional control method



Outline of proposed control method



Three Degree of Freedom of Robot Manipulator

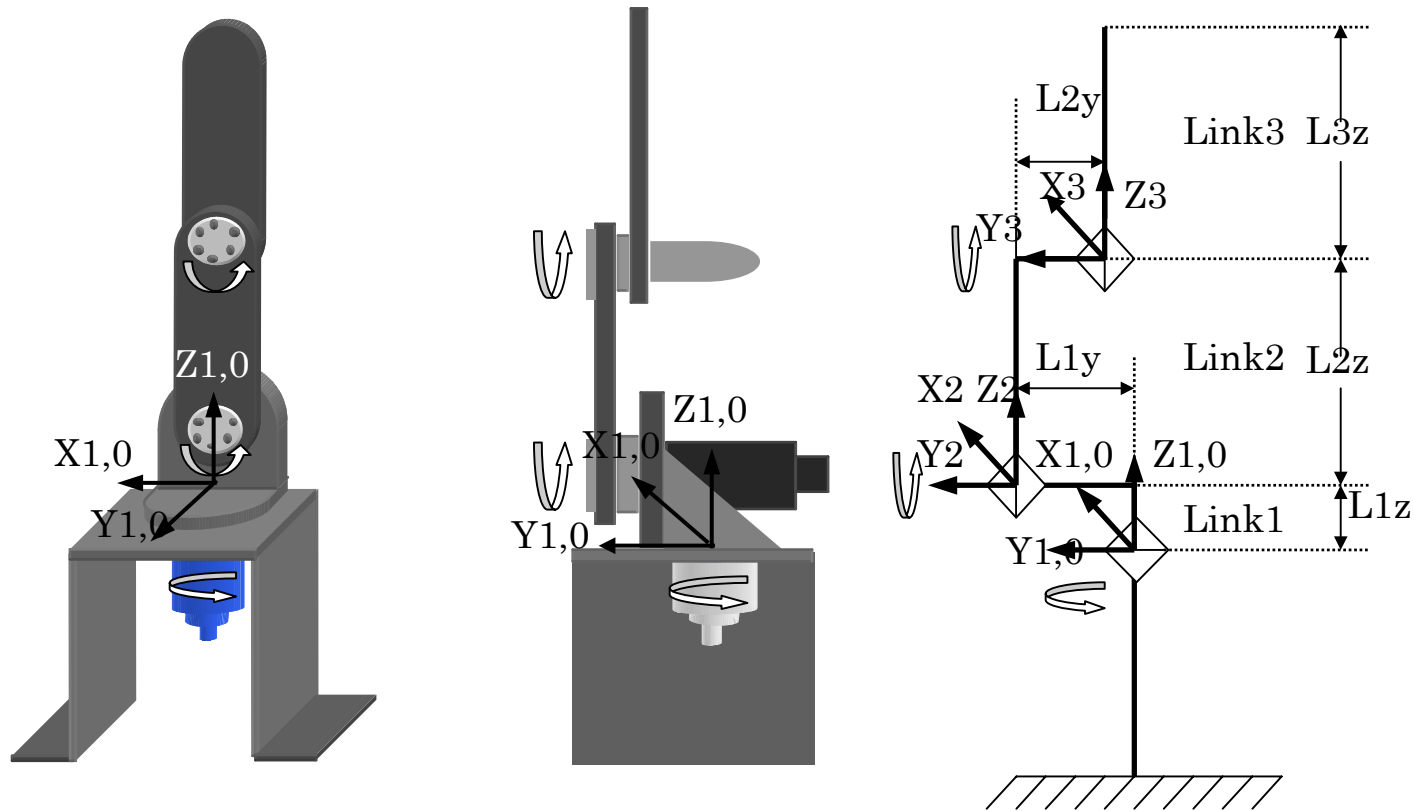
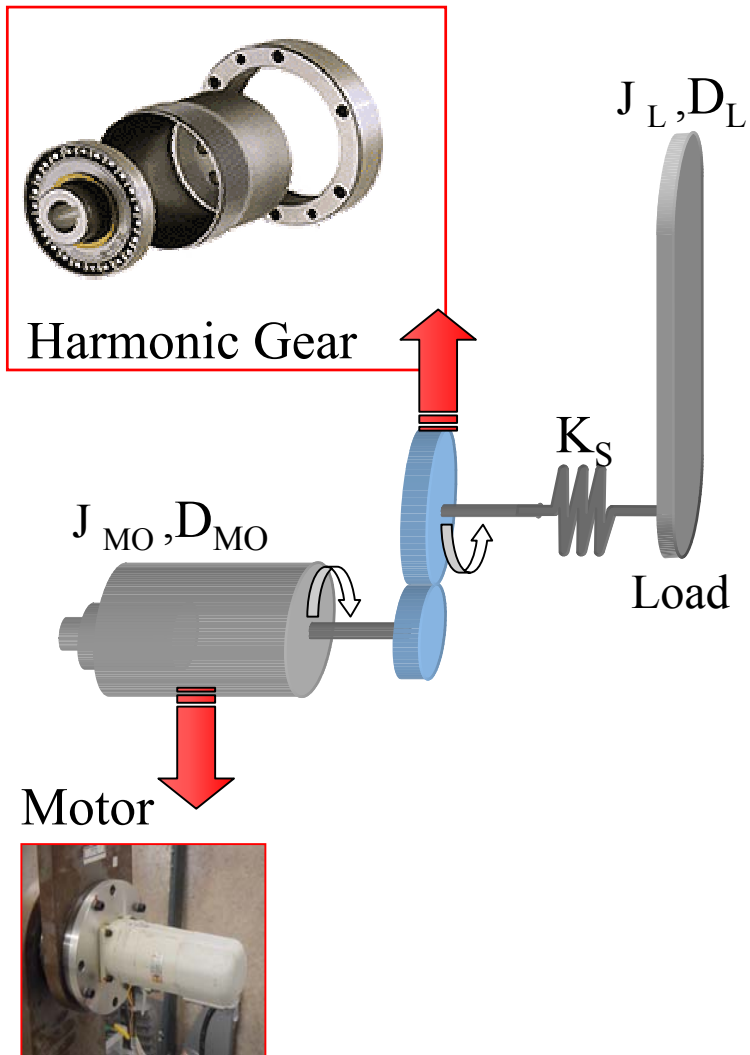


Figure 1: Description of the position and orientation of the end-effector frame

Two-Inertia Model



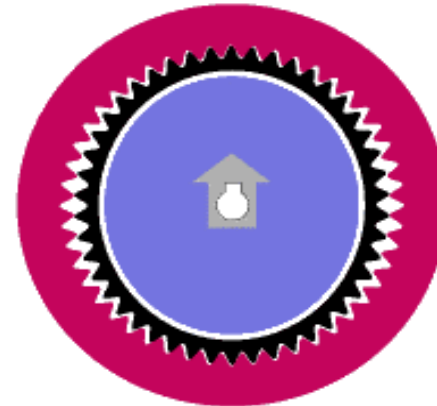
Advantages

Positioning accuracy and repeatability

Not back-drivable

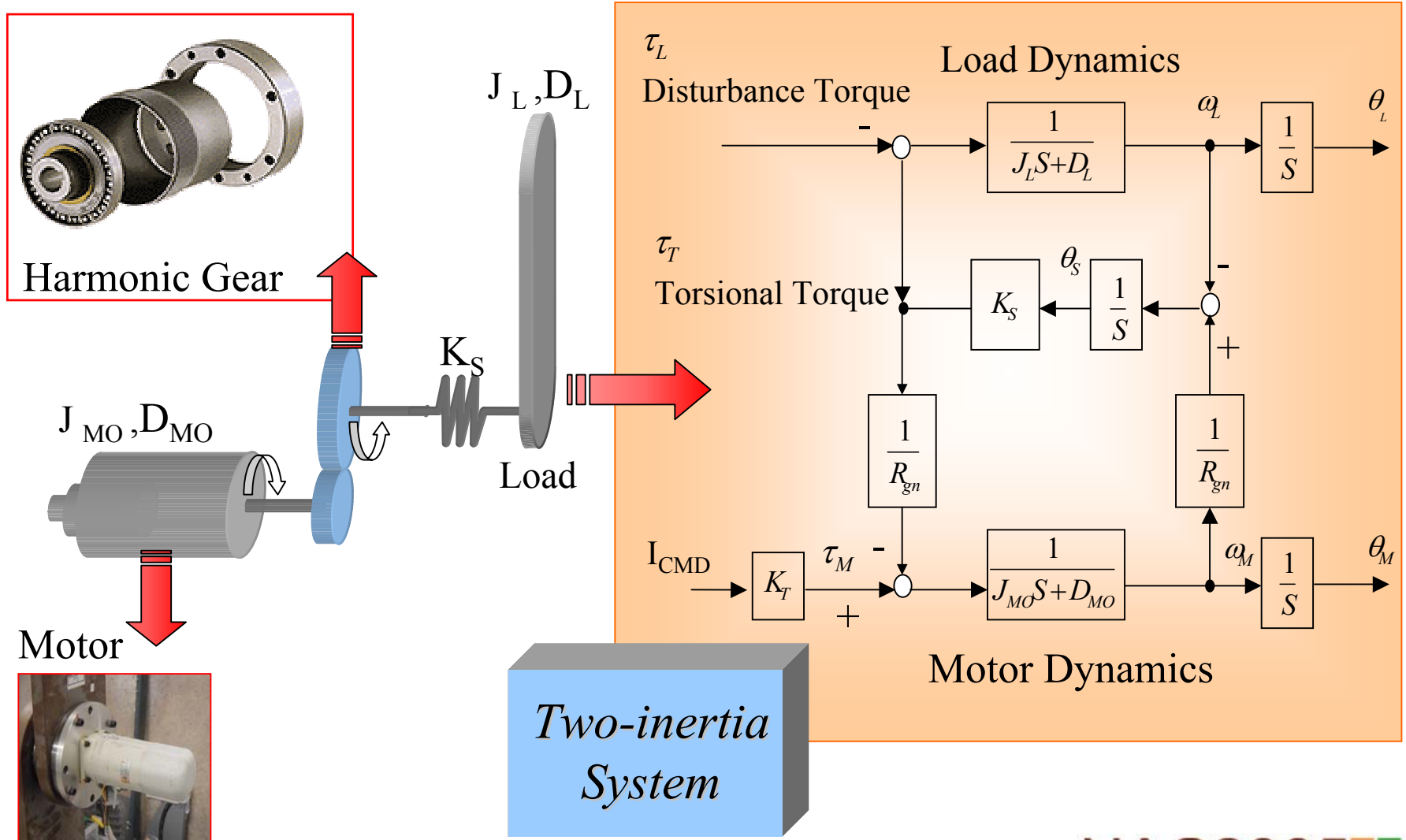
Zero-Backlash

High efficiency



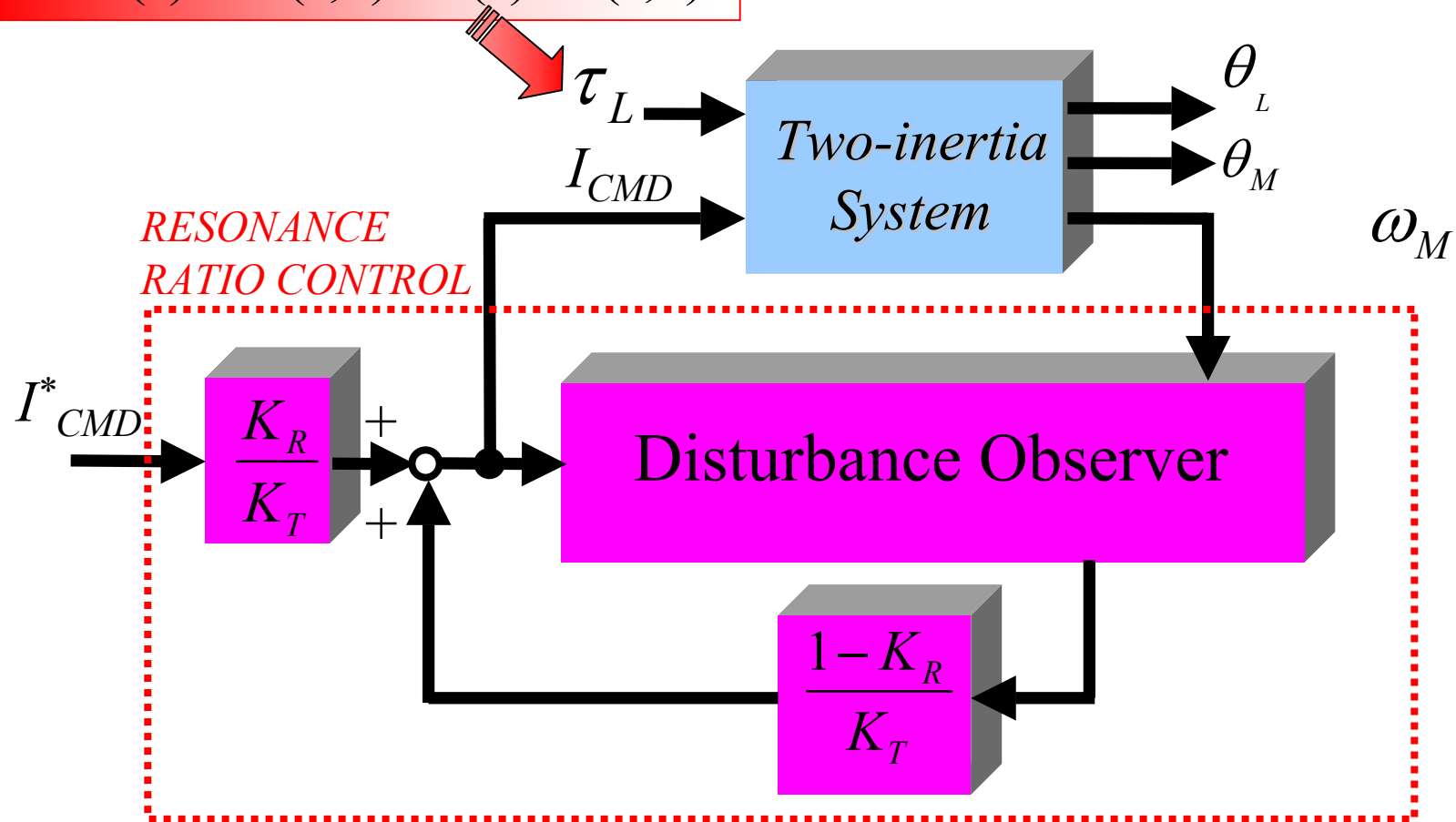
Motor in Motion Simulation

Two-Inertia Model



Resonance ratio control

$$\tau = D(\theta)\ddot{\theta} + h(\theta, \dot{\theta}) + C(\theta) + F(\theta, \dot{\theta})$$



Resonance ratio control and PI Speed Control

The resonance frequency of the new system is changed to

$$J_M = \frac{J_{MO}}{K_R} \quad \Rightarrow \quad \omega_r = \sqrt{\frac{K_s}{J_L} \left(1 + \frac{J_L K_R}{J_{MO} R g n^2} \right)}$$

Changed Resonance Frequency to

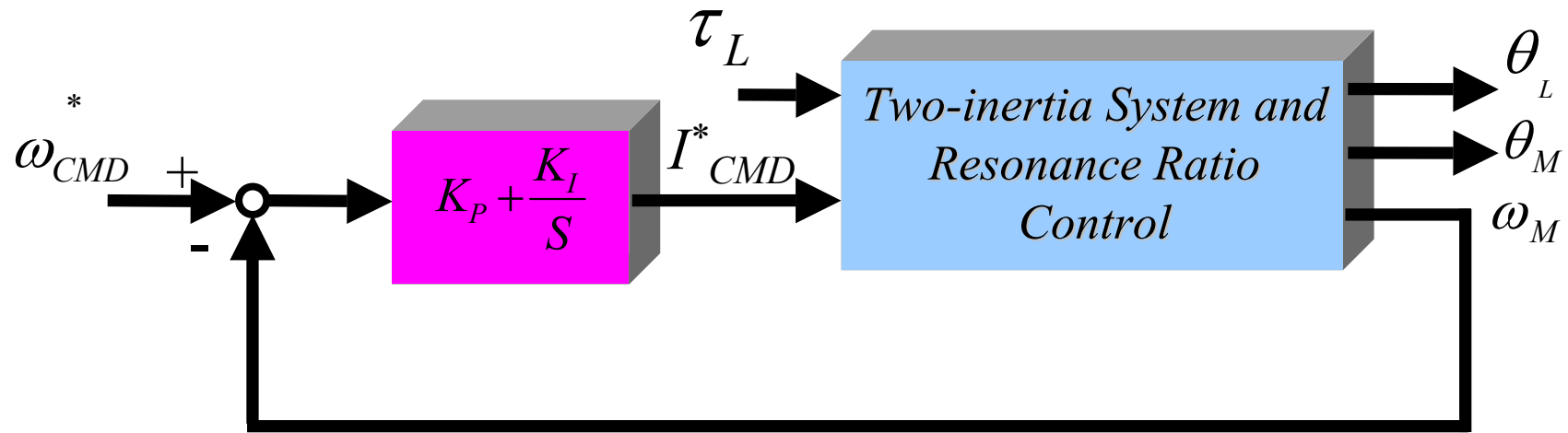
The resonance ratio can be controlled by K_R

$$H = \frac{\omega_r}{\omega_a} = \sqrt{1 + \frac{J_L K_R}{J_{MO} R g n^2}}$$

Optimal Resonance Ratio

Determined by CDM method

Resonance ratio control and PI Speed Control



Resonance ratio control and PI Speed Control

Manabe's Model Polynomial

$$\tau = \frac{a_1}{a_0} = \frac{K_p}{K_I} = \frac{1}{\beta} \quad \gamma_1 = \frac{a_1^2}{a_0 a_2} = 2.5, \quad \gamma_2 = \frac{a_2^2}{a_1 a_3} = 2, \quad \gamma_3 = \frac{a_3^2}{a_2 a_4} = 2$$

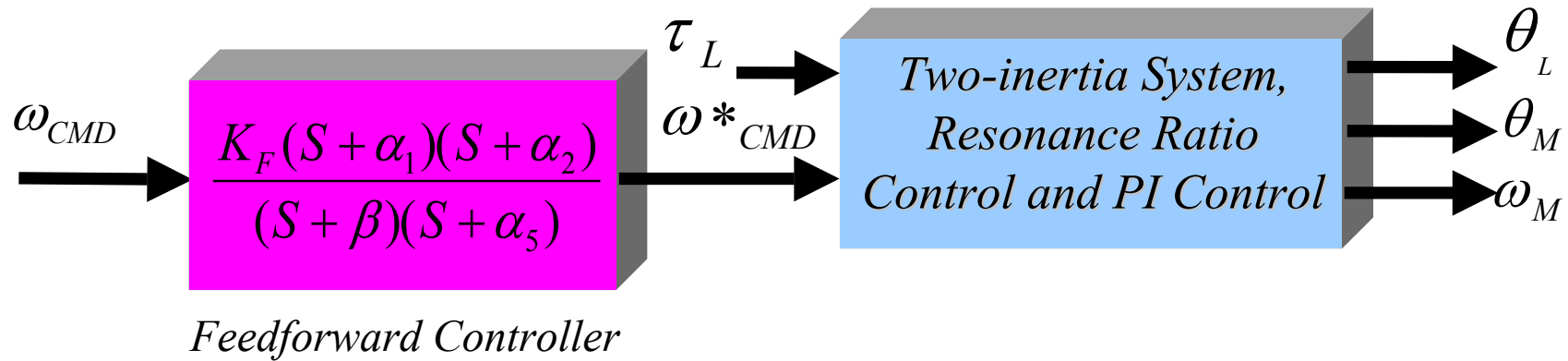
K_I, K_P and K_R Determined by CDM method

$$H = \frac{4\sqrt{5}}{5}, \quad K_I = \frac{4K_S}{11K_T R g n^2}, \quad K_P = \sqrt{\frac{(2 \times 16^2 + 40J_L)K_S}{11^2 K_T^2 R g n^4}}$$

Optimal Resonance Ratio

$$K_R = \frac{H^2 - 1}{R_O}$$

Feedforward Compensator

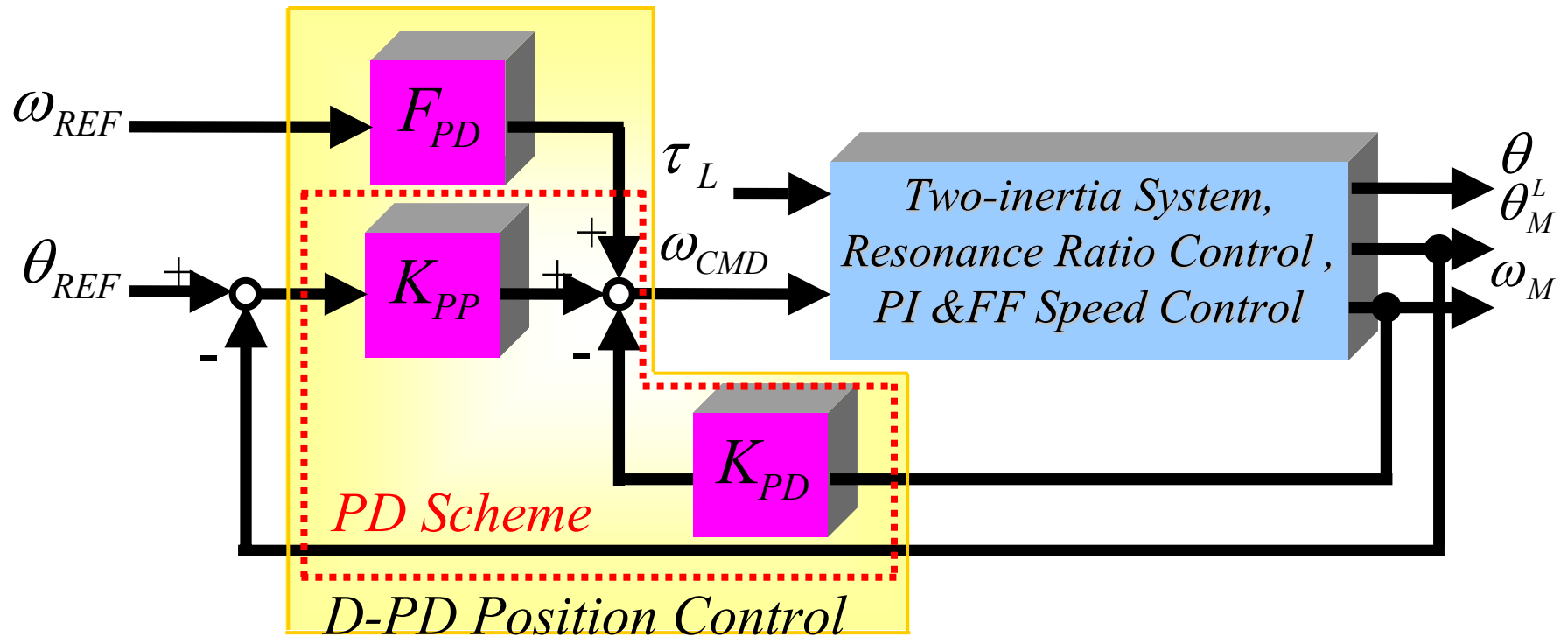


Eliminate effects of
dominant poles $-\alpha_1, -\alpha_2$



High Speed Response

D-PD Position Control



D-PD Position Control

Manabe's Model Polynomial

$$\tau = \frac{aa_1}{aa_0} = \frac{K_P}{K_I} = \frac{1}{\beta} \quad \text{Equivalent time constant}$$

$$\gamma_1 = \frac{aa_1^2}{aa_0aa_2} = 2.5 \quad \text{Stability index}$$

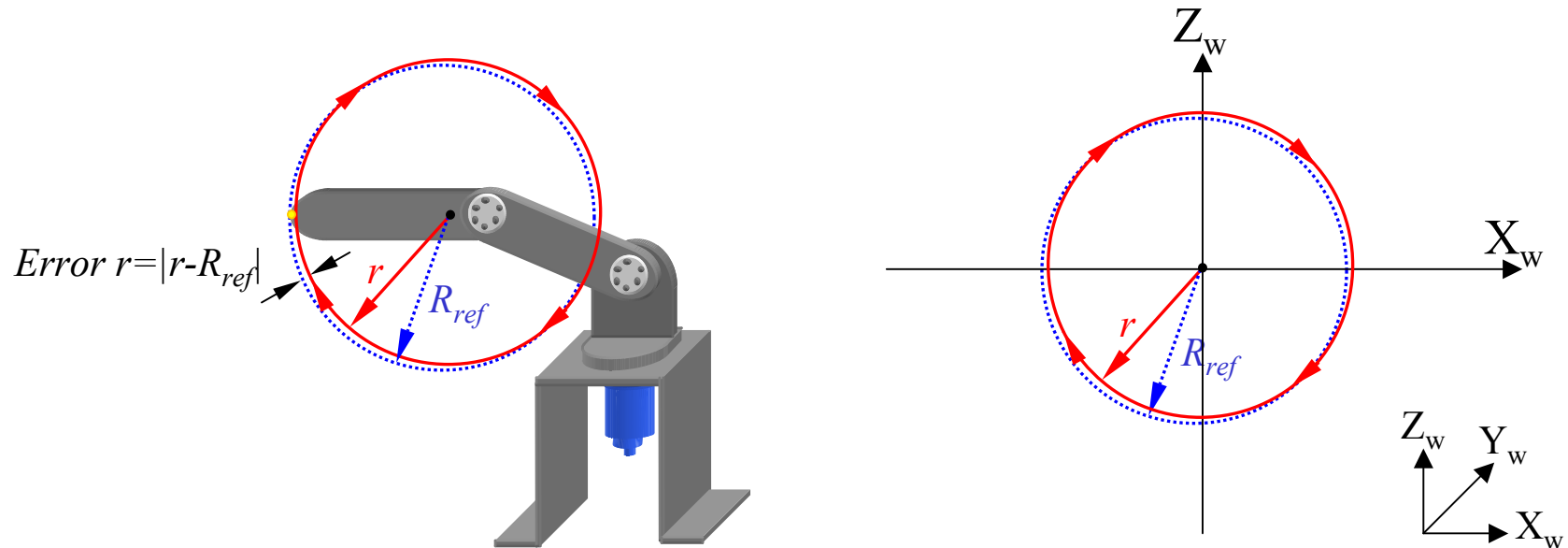
K_{PD} and K_{PP} Determined by CDM method

$$\begin{bmatrix} K_{PD} \\ K_{PP} \end{bmatrix} = \begin{bmatrix} K_{add}K_S & K_{add}D_L - \frac{K_S K_{add}}{\beta} \\ K_{add}K_S - K_{add}D_L\beta\gamma_1 & K_{add}D_L - K_{add}J_L\beta\gamma_1 \end{bmatrix}^{-1} \begin{bmatrix} -\alpha_3\alpha_4\alpha_5J_L \\ \alpha_3\alpha_4J_L\beta\gamma_1 + \alpha_4\alpha_5J_L\beta\gamma_1 + \alpha_3\alpha_5J_L\beta\gamma_1 - \alpha_3\alpha_4\alpha_5J_L \end{bmatrix}$$

F_{PD} Determined by arbitrary zero $-z_1$

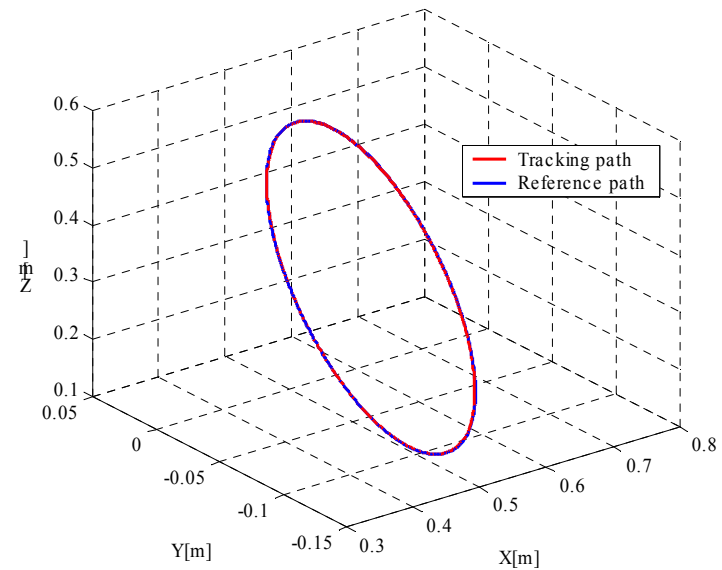
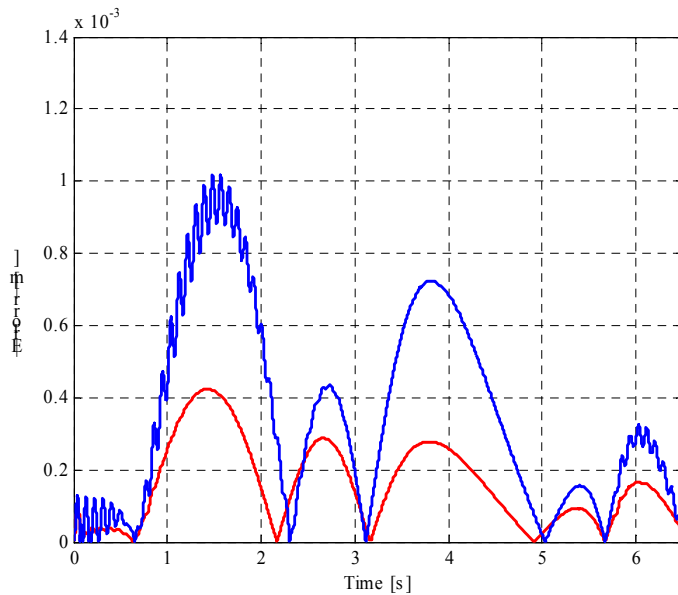
$$F_{pd} = \frac{K_{pp}}{z_1}$$

Simulation Results



The three-dimensional circle trajectory path which rotating angle from X-axis 45° and from Z-axis 45°

Simulation Results



<i>Controller</i>	<i>Maximum Error</i>	<i>Area of Error</i>
<i>Resonance Ratio Control , PI & FF Speed Control and D-PD Position Control</i>	<i>0.42 mm</i>	<i>0.0010 m-s</i>
<i>PI Speed Control and P Position Control</i>	<i>1.10mm</i>	<i>0.0023 m-s</i>

Conclusion

1. Vibration Suppression : Resonance Ratio Control and PI Speed Controller

2. Improve speed response: Feedforward control

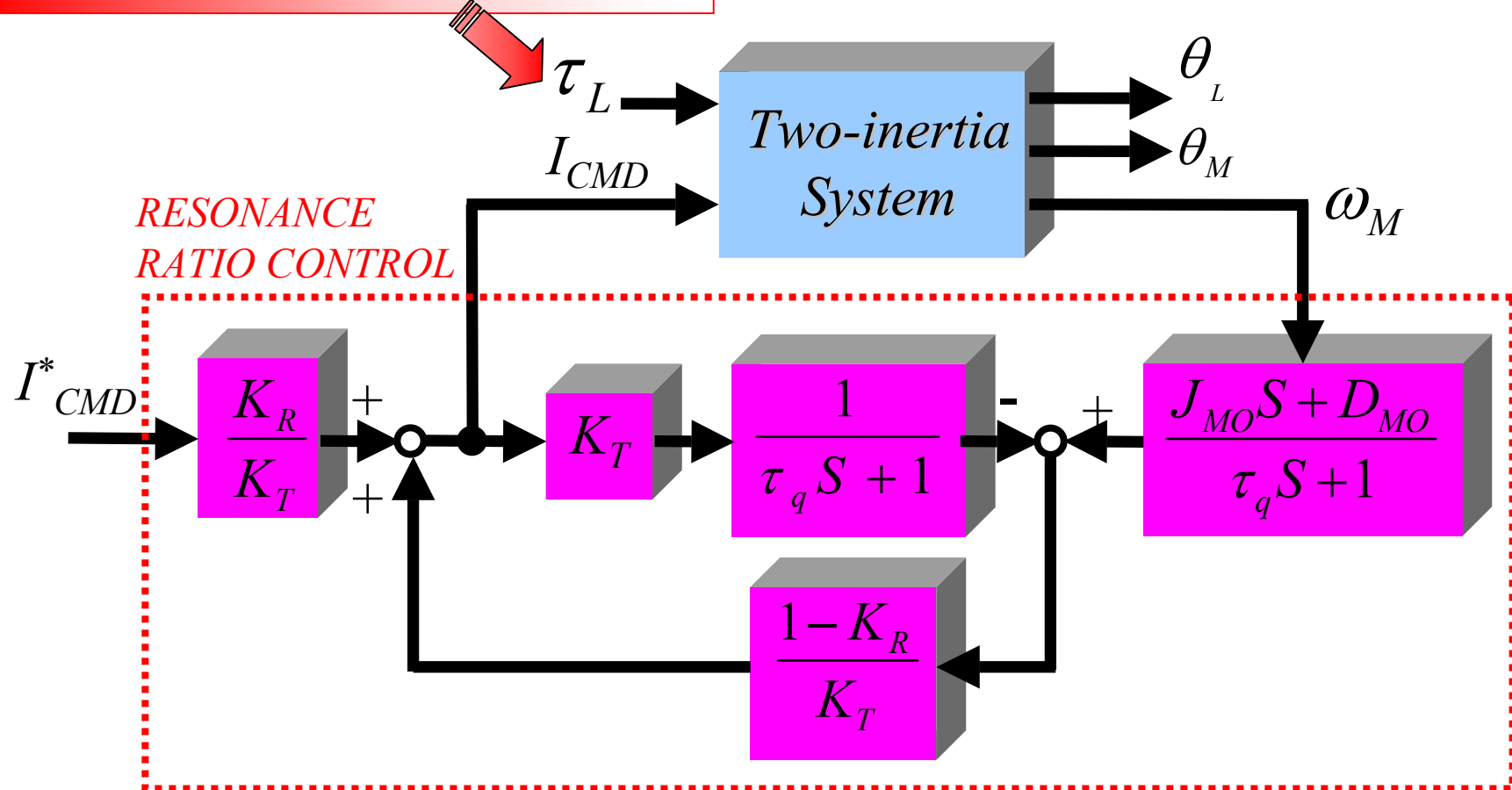
3. Reduce the state error of position control : D-PD Position Controller

4. All of this controller designed by CDM method, considering from stability index and rise time.

Thank you

Resonance ratio control

$$\tau = D(\theta)\ddot{\theta} + h(\theta, \dot{\theta}) + C(\theta) + F(\theta, \dot{\theta})$$



Dynamic of Robot Manipulator

$$\tau = \mathbf{D}(\theta)\ddot{\theta} + \mathbf{h}(\theta, \dot{\theta}) + \mathbf{C}(\theta) + \mathbf{F}(\theta, \dot{\theta})$$

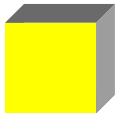
$$\begin{bmatrix} \tau_1 \\ \tau_2 \\ \tau_3 \end{bmatrix} = \begin{bmatrix} D_{11} & D_{12} & D_{13} \\ D_{21} & D_{22} & D_{23} \\ D_{31} & D_{32} & D_{33} \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{bmatrix} + \begin{bmatrix} h_{11} & h_{12} & h_{13} \\ h_{21} & h_{22} & h_{23} \\ h_{31} & h_{32} & h_{33} \end{bmatrix} \begin{bmatrix} \dot{\theta}_1^2 \\ \dot{\theta}_2^2 \\ \dot{\theta}_3^2 \end{bmatrix} + \begin{bmatrix} hc_{11} & hc_{12} & hc_{13} \\ hc_{21} & hc_{22} & hc_{23} \\ hc_{31} & hc_{32} & hc_{33} \end{bmatrix} \begin{bmatrix} \dot{\theta}_1 \dot{\theta}_2 \\ \dot{\theta}_2 \dot{\theta}_3 \\ \dot{\theta}_1 \dot{\theta}_3 \end{bmatrix} + \begin{bmatrix} C_1 \\ C_2 \\ C_3 \end{bmatrix} + \begin{bmatrix} F_1 \\ F_2 \\ F_3 \end{bmatrix}$$



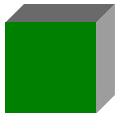
Acceleration-relate symmetric matrix term



Nonlinear Coriolis and centrifugal force vector term



Gravity loading force vector term



Coulomb and viscous friction term